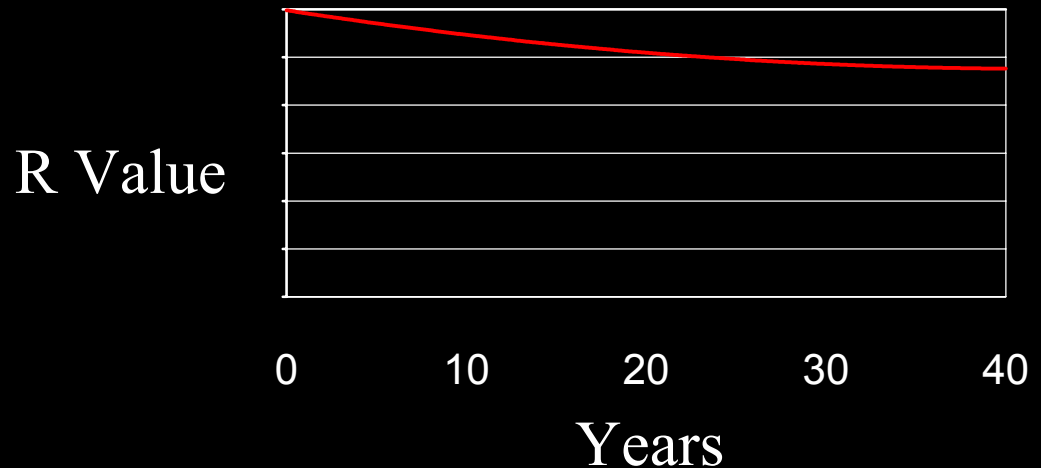


# Experimental Progress In Predicting VIP Life-Long Performance



# Vacuum Panel Designed Life

- Vacuum panel life **is not just predicted**
- The panel is **designed for a particular life**



# VIP Performance vs. Time

- Atmospheric gases such as oxygen and nitrogen permeates into the panel
- Water vapor permeates into the panel

# Focus of This Presentation

- Water vapor permeation into the VIP
  - ◆ Function of:
    - ★ Barrier properties
    - ★ Driving force (VIP exterior temperature and humidity)

# Review of Proposed Theory

- At the last symposium in Vancouver, I proposed a theoretical model of VIP moisture environment in an appliance wall
- This presentation will cover **measured** temperature and humidity inside appliance walls
- First lets briefly review the theory

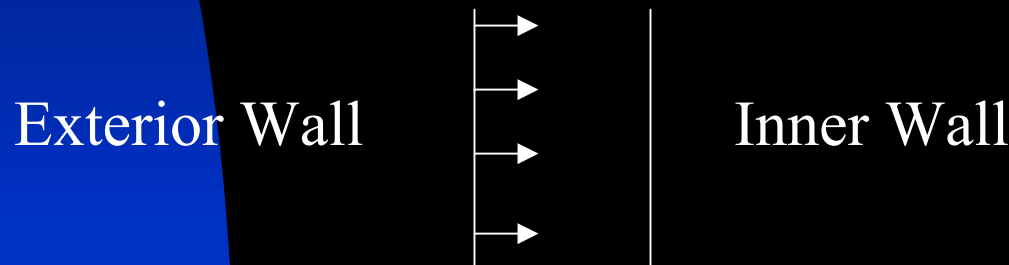
# Refrigerator/freezer Application

- A first assumption is that the vacuum insulation panel is in the same environment as the exterior of the refrigerator/freezer
- I propose that this assumption is not only conservative but in error

# The Special Case of a Refrigerator or Freezer

## Theory

- Virtually continuous operation over its life
- The cooling system effectively provides a moisture pump to move the moisture that gets into the wall to the cold inner wall



# Refrigerator/freezer

## Theory

- Moisture may enter the refrigerator wall and move to the freezer inner wall
  - ◆ Coldest spot (lowest energy)
- Doors are of course handled separately



# Approximate Moisture Permeance

- Permeance is a measure of the water transmission rate of a material
  - ◆ Grams of water per hour square meter, millimeter of mercury vapor pressure difference
- Or WVTR (water vapor transmission rate – grams/sq.meter Day)
- Steel\* = 0 WVTR
- \*Note there are penetrations
  - ◆ However, manufacturers are careful to maintain a good moisture barrier

# Approximate Moisture Permeance

- Barrier film = 0.0026 WVTR
- Urethane > 5000 WVTR

# What Is the Vacuum Panel Environment?

- Relative humidity and temperature
  - ◆ Vapor pressure results from both relative humidity and temperature
    - ★ Note: temperature is still important by itself since barrier performance is a function of temperature and vapor pressure

# What Is the Relative Humidity and Temperature in the Wall?

## Theory

- If moisture gets into the wall it rapidly moves to the cold interior wall
- The maximum vapor pressure in the wall will be the vapor pressure of 100% relative humidity at the temperature of the cold inner wall

# Vapor Pressure at 100% Relative Humidity

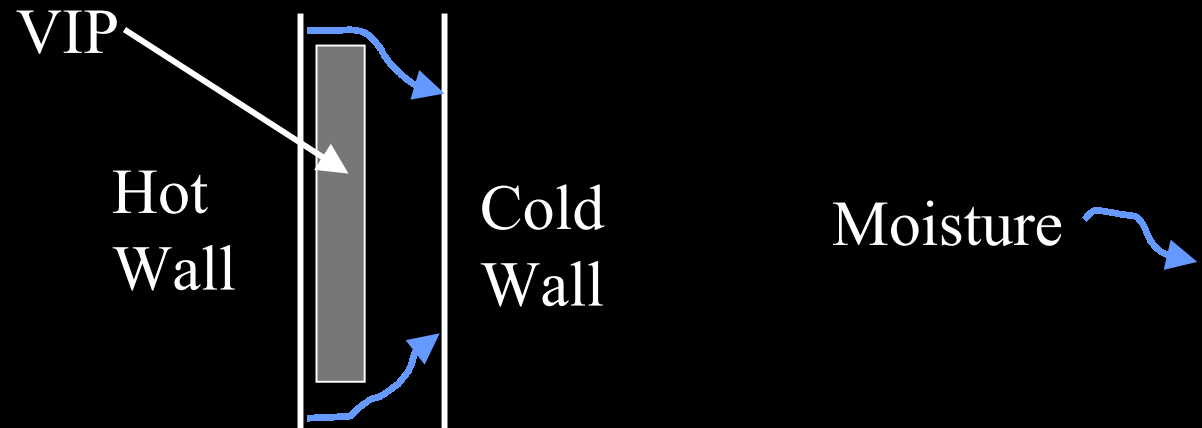
- Refrigerator at  $3.3^{\circ}\text{C}$  = 5.58 mm Hg
- Freezer at  $-23^{\circ}\text{C}$  = 0.55 mm Hg
- The wall cavity will be at equilibrium with one of the above

**Theory**

# Inside the Wall at the Hot Wall

## Theory

- The vapor pressure must be equal to the cold wall
  - ◆ Result of the rapid diffusion of water through urethane foam



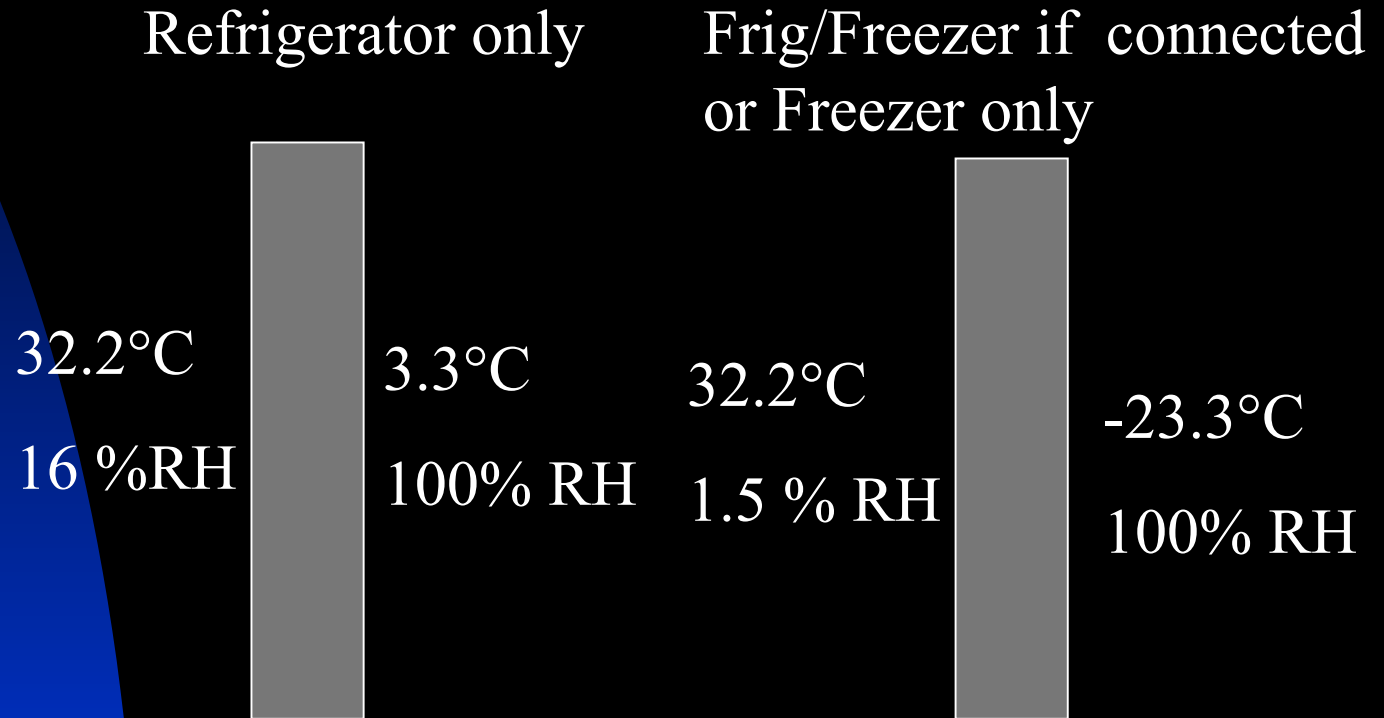
# Inside the Wall at the Hot Wall

- A relative humidity inside the wall at the hot wall can be calculated from the vapor pressure
  - ◆ Refrigerator
    - ★ If exterior temperature is 21.1°C, RH = 31%
    - ★ If exterior temperature is 32.2°C, RH = 16%
  - ◆ Freezer
    - ★ If exterior temperature is 21.1°C, RH = 3%
    - ★ If exterior temperature is 32.2°C, RH = 1.5%

Theory

# Thus, Vacuum Panel Environment for Moisture Is:

**Theory**



Half barrier film area at hot wall condition and half at the cold wall condition



# The Previous Slides Were Theory - Now What Does the Test Data Show

## Test Unit

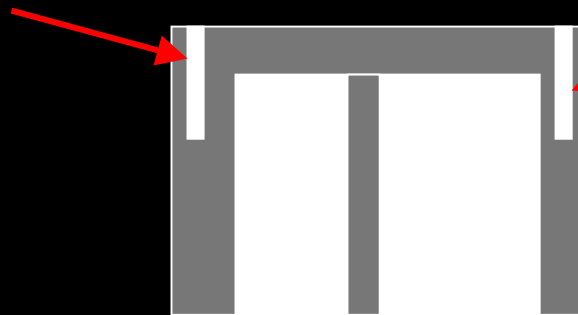
- Actual temperature and humidity were measured.
- Refrigerator/Freezer Test Unit
  - A 21 cubic foot ( 0.59 cubic meters) side by side refrigerator that had been operating in a residential house for 22 years was selected
  - Freezer wall thickness = 38 mm
  - Refrigerator wall thickness = 38 mm

# Temperature and Humidity Measurement

## Measurement

- A combined temperature/humidity probe was used.
- Holes were drilled into the refrigerator and freezer sidewalls from the back wall. The 9 mm holes were drilled parallel the sidewalls.

Drilled  
hole for  
probe



Drilled  
hole for  
probe

# Test Results

## Test Results

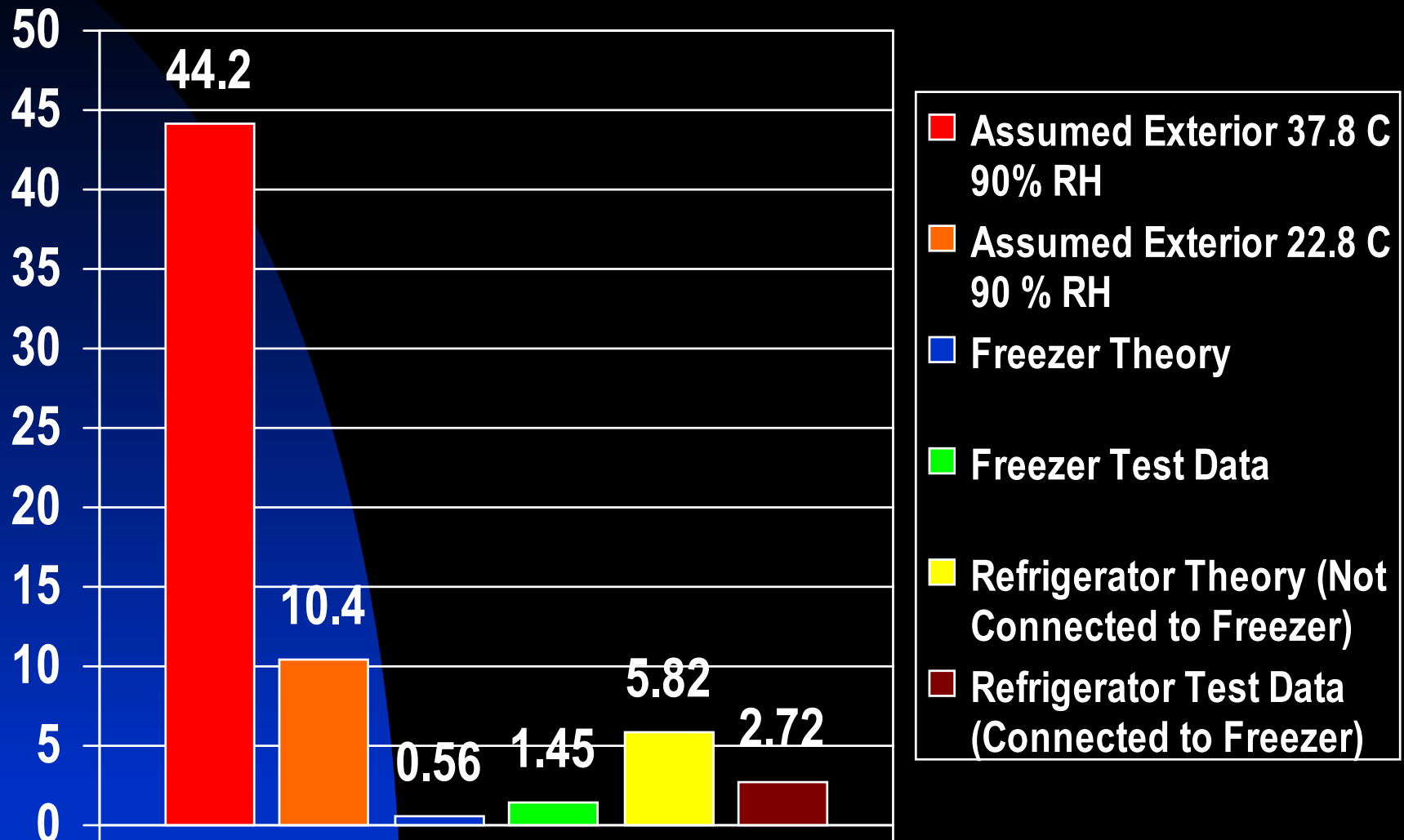
- Inside freezer wall: 8.9 C and 17% RH or vapor pressure 1.45 mm Hg

(Theory says 0.56 mm Hg)

- Inside refrigerator wall (connected to freezer): 14.4 C and 22% RH or vapor pressure 2.72 mm Hg

(Theory says same as freezer if refrigerator and freezer are connected: 0.56 mm Hg or 5.81 mm Hg if not connected)

# Water Vapor Pressure (mm Hg)



# Panel Environment

- **Far** less severe than the original assumption of the exterior room conditions
- If refrigerator is connected to the freezer, then there is less vapor pressure than a refrigerator by itself

# What Does This Mean - Example

- VIP 508 x 584 x 25.4 mm
  - ◆ Metalized barrier film
  - ◆ 20 year life
  - ◆ Freezer wall
    - ★ Requires about 5 grams of desiccant
  - ◆ Refrigerator wall connected to freezer
    - ★ Requires about 10 grams of desiccant
  - ◆ Refrigerator wall NOT connected to freezer
    - ★ Requires about 25 grams of desiccant

# Now The Other Part of The Problem

- Moisture pickup in VIP before the appliance is turned on
  - ◆ Storage before shipment to appliance manufacturer
  - ◆ During shipment to appliance manufacturer
  - ◆ Storage before manufacture
  - ◆ Storage after manufacture
  - ◆ Shipment to distributor
  - ◆ Storage at distributor before consumer begins use

# Calculation Before the Appliance is Turned On

- A bin method of calculation can be used
  - ◆ Time period selected (hour, daytime, nighttime, week, month, etc.)
  - ◆ Temperature and humidity defined for the time period
    - ★ Climate data is available on the internet or from sources such as American Society of Heating Refrigeration and Air-conditioning Engineers (ASHRAE)
  - ◆ Average water vapor pressure for each time period estimated
  - ◆ From the barrier film water vapor transmission rate versus vapor pressure: calculate required desiccant for the time period



# Example – Exposure Before Appliance Is Turned On

- New Orleans, Louisiana, U.S.A. was selected as a sever location
  - ◆ Along the Gulf Coast of the United States – very hot and humid
  - ◆ Sever conditions for New Orleans were assumed

# New Orleans Example

New Orleans		Vacuum Panel 508 x 584 x 25.4 mm		
Month	Temperature	% RH	Vapor Pressure (mm Hg)	Grams of Desiccant
May	Day 29 C ( 84 F)	90	26.9	0.41
	Night 23 C (73 F)	90	18.8	0.24
June	Day 32 C (90 F)	90	32.5	0.55
	Night 26.6 C (80 F)	90	23.6	0.34
July	Day 33 C (92 F)	90	34.5	0.61
	Night 27.8 C (82 F)	90	25.1	0.37
August	Day 33 C (92 F)	90	34.5	0.61
	Night 27.8 C (82 F)	90	25.1	0.37
September	Day 29.4 C (85 F)	90	27.7	0.43
	Night 23.9 C (75 F)	90	20.1	0.26
October	Day 25.5 C (78 F)	90	22.1	0.30
	Night 18.9 C (66 F)	90	14.7	0.17
<b>Total</b>				<b>4.66</b>

# Next Steps – Further Validation

- More temperature-humidity data in refrigerator and freezer walls should be collected
- Moisture gain in the desiccant **before the unit is turn on** should be collected
  - ◆ Desiccant weighed before inserted in the VIP and then appliance dissected after normal ship and storage to determine the final desiccant weight gain

# Conclusions:

- The moisture environment in refrigerator and freezer walls is much less severe than the room ambient
  - ◆ Small amounts of desiccant required for very long life
- The environment and moisture pickup before the appliance is turned on should **not** be ignored